AMENDMENTS TO THE SPECIFICATION:

Please add the following ***new*** paragraph on page 1, between lines 2 and 3:

CROSS-REFERENCE TO RELATED APPLICATIONS

This U.S. National stage application claims priority under 35 U.S.C. §119(a) to Japanese Patent Application No. 2004-203665, filed in Japan on July 9, 2004, the entire contents of which are hereby incorporated herein by reference.

Please replace paragraph [0002] with the following rewritten version:

[0002] As a conventional example of a fluid machine, ~~Patent Publication 1~~ Japanese Unexamined Patent Publication No. H6-288358 discloses a compressor having an eccentric rotation piston mechanism achieved by a cylinder having an annular cylinder chamber and an annular piston which is contained in the cylinder chamber to make eccentric rotation. The fluid machine compresses a refrigerant by making use of volumetric change in the cylinder chamber caused by the eccentric rotation of the piston.

~~Patent Publication 1: Japanese Unexamined Patent Publication No. H6-288358~~

Please replace the heading at page 1, line 15, with the following rewritten version:

SUMMARY OF THE INVENTION ~~DISCLOSURE OF THE INVENTION~~

Please replace paragraph [0006] with the following rewritten version:

[0006] Specifically, as shown in FIG. **1**, a first aspect of the present invention is directed to a rotary fluid machine including a rotation mechanism (**20**) including: a cylinder (**21**) having an annular cylinder chamber (**50**); an annular piston (**22**) which is contained in the cylinder chamber (**50**) to be eccentric to the cylinder (**21**) and divides the cylinder chamber (**50**) into an outer working chamber (**51**) and an inner working chamber (**52**); and a blade (**23**) which is arranged in the cylinder chamber (**50**) to divide each of the working chambers into a high

pressure region and a low pressure region, the cylinder (**21**) and to the piston (**22**) making relative rotations, wherein the width **T1** of the cylinder chamber (**50**) is varied along the circumference of the cylinder chamber (**50**) such that the gap between the wall surface of the cylinder (**21**) and the wall surface of the piston (**22**) is kept to a predetermined value during the rotations.

Please replace paragraph [0007] with the following rewritten version:

[0007] According to the first aspect of the present invention, when the rotation mechanism (**20**) is actuated, the cylinder (**21**) and the piston (**22**) make relative rotations. As a result, the volumes of the working chambers (**51**) and (**52**) vary to cause compression or expansion of a fluid. As the width **T1** of the cylinder chamber (**50**) is varied along the circumference thereof, the gap that occurs between the wall surface of the cylinder (**21**) and the wall surface of the piston (**22**) is reduced to a minimum.

Please replace paragraph [0008] with the following rewritten version:

[0008] A second aspect of the present invention is directed to a rotary fluid machine including a rotation mechanism (**20**) including: a cylinder (**21**) having an annular cylinder chamber (**50**); an annular piston (**22**) which is contained in the cylinder chamber (**50**) to be eccentric to the cylinder (**21**) and divides the cylinder chamber (**50**) into an outer working chamber (**51**) and an inner working chamber (**52**); and a blade (**23**) which is arranged in the cylinder chamber (**50**) to divide each of the working chambers into a high pressure region and a low pressure region, the cylinder (**21**) and the piston (**22**) make relative rotations without spinning by themselves, wherein the width **T2** of the piston (**22**) is varied along the circumference of the piston (**22**) such that the gap between the wall surface of the cylinder (**21**) and the wall surface of the piston (**22**) is kept to a predetermined value during the rotations.

Please replace paragraph [0009] with the following rewritten version:

[0009] According to the second aspect of the present invention, when the rotation mechanism (**20**) is actuated, the cylinder (**21**) and the piston (**22**) make relative rotations. As a result, the volumes of the working chambers (**51**) and (**52**) vary to cause compression or expansion of a fluid. As the width **T2** of the piston (**22**) is varied along the circumference of the piston (**22**), the gap that occurs between the wall surface of the cylinder (**21**) and the wall surface of the piston (**22**) is reduced to a minimum.

Please replace paragraph [0010] with the following rewritten version:

[0010] According to a third aspect of the present invention related to the second aspect of the present invention, the width **T1** of the cylinder chamber (**50**) is varied along the circumference of the cylinder chamber (**50**) such that the gap between the wall surface of the cylinder (**21**) and the wall surface of the piston (**22**) is kept to a predetermined value during the rotations.

Please replace paragraph [0011] with the following rewritten version:

[0011] According to the third aspect of the present invention, the width **T1** of the cylinder chamber (**50**) is varied along the circumference of the cylinder chamber (**50**) and the width **T2** of the piston (**22**) is varied along the circumference of the piston (**22**). Therefore, the gap that occurs between the wall surface of the cylinder (**21**) and the wall surface of the piston (**22**) is reduced to a minimum.

Please replace paragraph [0012] with the following rewritten version:

[0012] According to a fourth aspect of the present invention related to the first or third aspect of the present invention, regarding the center line of the blade (**23**) as a starting point of the circumference of the cylinder chamber (**50**), the width **T1** of part of the cylinder chamber (**50**) ranging from the starting point to a point at a rotation angle of 180° from the starting point is small and the width **T1** of the other part of the cylinder chamber (**50**) ranging from the 180° point to a point at a rotation angle less than 360° from the starting point is small.

Please replace paragraph [0013] with the following rewritten version:

[0013] According to the fourth aspect of the present invention, the gap that occurs between the wall surface of the cylinder (**21**) and the wall surface of the piston (**22**) is reduced to a minimum with higher reliability.

Please replace paragraph [0014] with the following rewritten version:

[0014] According to a fifth aspect of the present invention, the center of the inner circumference of the cylinder chamber (**50**) is deviated from the center of the outer circumference of the cylinder chamber (**50**) when viewed in plan.

Please replace paragraph [0015] with the following rewritten version:

[0015] According to the fifth aspect of the present invention, the cylinder (**21**) is fabricated easily by merely deviating the inner circumference center and the outer circumference center of the cylinder chamber (**50**) from each other.

Please replace paragraph [0016] with the following rewritten version:

[0016] According to a sixth aspect of the present invention related to the first or third aspect of the present invention, the cylinder chamber (**50**) is divided into four regions along the circumference thereof such that the cylinder chamber (**50**) has wide regions (**Z1**, **Z3**) and narrow regions (**Z2**, **Z4**) formed in a continuous and alternate manner.

Please replace paragraph [0017] with the following rewritten version:

[0017] According to the sixth aspect of the present invention, the gap that occurs between the wall surface of the cylinder (**21**) and the wall surface of the piston (**22**) is surely reduced to a minimum at any time during the relative rotations by the cylinder (**21**) and the piston (**22**).

Please replace paragraph [0018] with the following rewritten version:

[0018] According to a seventh aspect of the present invention related to the second or third aspect of the present invention, the blade (**23**) and the piston (**22**) make relative swings at a predetermined swing center and regarding the swing center of the blade (**23**) and the piston (**22**) as a starting point of the circumference of the piston (**22**), the width **T2** of part of the piston (**22**) ranging from the starting point to a point at a rotation angle of 180° from the starting point is small and the width **T2** of the other part of the piston (**22**) ranging from the 180° point to a point at a rotation angle of 360° from the starting point is large.

Please replace paragraph [0019] with the following rewritten version:

[0019] According to the seventh aspect of the present invention, the gap that occurs between the wall surface of the cylinder (**21**) and the wall surface of the piston (**22**) is reduced to a minimum with higher reliability.

Please replace paragraph [0020] with the following rewritten version:

[0020] According to an eighth aspect of the present invention related to the seventh aspect of the present invention, the center of the inner circumference of the piston (**22**) is deviated from the center of the outer circumference of the piston (**22**) when viewed in plan.

Please replace paragraph [0021] with the following rewritten version:

[0021] According to the eighth aspect of the present invention, the piston (**22**) is fabricated easily by merely deviating the inner circumference center and the outer circumference center of the piston (**22**) from each other.

Please replace paragraph [0022] with the following rewritten version:

[0022] According to a ninth aspect of the present invention related to the second or third aspects of the present invention, the blade (**23**) and the piston (**22**) make relative swings at a predetermined swing center and the piston (**22**) is divided into four regions along the circumference thereof such that the piston (**22**) has narrow regions (**W1, W3**) and wide regions (**W2, W4**) formed in a continuous and alternate manner.

Please replace paragraph [0023] with the following rewritten version:

[0023] According to the ninth aspect of the present invention, the gap that occurs between the wall surface of the cylinder (**21**) and the wall surface of the piston (**22**) is surely reduced to a minimum at any time during the relative rotations by the cylinder (**21**) and the piston (**22**).

Please replace paragraph [0024] with the following rewritten version:

[0024] According to a tenth aspect of the present invention related to the first aspect of the present invention, part of the annular piston (**22**) of the rotation mechanism (**20**) is cut off such that the piston (**22**) is C-shaped, the blade (**23**) of the rotation mechanism (**20**) extends from the inner wall surface to the outer wall surface of the cylinder chamber (**50**) and passes through the cut-off portion of the piston (**22**) and a swing bushing is provided in the cut-off portion of the piston (**22**) to contact the piston (**22**) and the blade (**23**) via the surfaces thereof such that the blade (**23**) freely reciprocates and the blade (**23**) and the piston (**22**) make relative swings.

Please replace paragraph [0025] with the following rewritten version:

[0025] According to the tenth aspect of the present invention, the blade (**23**) reciprocates through the swing bushing (**27**) and the blade (**23**) swings together with the swing bushing (**27**) relative to the piston (**22**). Therefore, the cylinder (**21**) and the piston (**22**) make relative swings and rotations, whereby the rotation mechanism (**20**) achieves predetermined work such as compression.

Please replace paragraph [0027] with the following rewritten version:

[0027] According to the fourth aspect of the present invention, the width **T1** of part of the cylinder chamber (**50**) ranging from the starting point to a point at a rotation angle of 180° from the starting point is large and the width **T1** of the other part of the cylinder chamber (**50**) ranging from the 180° point to a point at a rotation less than 360° from the starting point is

small. Further, according to the seventh aspect of the present invention, the width **T2** of part of the piston (**22**) ranging from the starting point to a point at a rotation angle of 180° from the starting point is small and the width **T2** of the other part of the piston (**22**) ranging from the 180° point to a point at a rotation angle of 360° from the starting point is large. Therefore, the refrigerant leakage is surely prevented at any time during a single rotation. This brings about an improvement in efficiency with reliability.

Please replace paragraph [0028] with the following rewritten version:

[0028] According to the fifth aspect of the present invention, the center of the inner circumference of the cylinder chamber (**50**) is deviated from the center of the outer circumference of the cylinder chamber (**50**) when viewed in plan. Further, according to the eighth aspect of the present invention, the center of the inner circumference of the piston (**22**) is deviated from the center of the outer circumference of the piston (**22**) when viewed in plan. Therefore, the width **T1** of the cylinder chamber (**50**) is easily varied, and so is the width **T2** of the piston (**22**).

Please replace paragraph [0029] with the following rewritten version:

[0029] According to the sixth aspect of the present invention, the cylinder chamber (**50**) is divided into four regions along the circumference thereof such that the cylinder chamber (**50**) has wide regions (**Z1**, **Z3**) and narrow regions (**Z2**, **Z4**) formed in a continuous and alternate manner. Further, according to the ninth aspect of the present invention, the piston (**22**) is divided into four regions along the circumference thereof such that the piston (**22**) has narrow regions (**W1**, **W3**) and wide regions (**W2**, **W4**) formed in a continuous and alternate manner. Therefore, the gap that occurs between the wall surface of the cylinder (**21**) and the wall surface of the piston (**22**) is surely reduced to a minimum at any time during the relative rotations by the cylinder (**21**) and the piston (**22**).

Please replace paragraph [0030] with the following rewritten version:

[0030] According to the tenth aspect of the present invention, a swing bushing (**27**) is provided as a connector for connecting the piston (**22**) and the blade (**23**) and substantially contacts the piston (**22**) and the blade (**23**) via the surfaces thereof. Therefore, the piston (**22**) and the blade (**23**) are prevented from wearing away and seizing up at the contacting parts.

Please remove the heading at page 8, line 5 as follows:

~~**BRIEF EXPLANATION OF REFERENCE NUMERALS**~~

Please remove the paragraph on page 8, line 6 as follows:

~~[0034] 1 Compressor~~
~~10 Casing~~
~~20 Compressor mechanism~~
~~21 Cylinder~~
~~22 Piston~~
~~23 Blade~~
~~24 Outer cylinder~~
~~25 Inner cylinder~~
~~27 Swing bushing~~
~~30 Motor (drive mechanism)~~
~~33 Drive shaft~~
~~50 Cylinder chamber~~
~~51 Outer compression chamber~~
~~52 Inner compression chamber~~

Please replace the heading at page 8, line 20, with the following rewritten version:

~~BEST MODES FOR CARRYING OUT~~ DETAILED DESCRIPTION OF THE INVENTION

Please replace the heading at page 25, line 1, with the following rewritten version:

WHAT IS CLAIMED IS: ~~CLAIMS~~